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Manipal Institute of Technology, Manipal

(A Constituent Institute of Manipal University)



VI SEMESTER B.TECH (AUTOMOBILE ENGINEERING)

END SEMESTER EXAMINATIONS, MAY/JUNE 2016

SUBJECT: **COMBUSTION AND HEAT TRANSFER [AAE 352]**

REVISED CREDIT SYSTEM

Time: 3 Hours

MAX. MARKS: 50

Instructions to Candidates:

- ❖ Answer **ANY FIVE FULL** the questions.
- ❖ Use of Heat and Mass transfer Data Hand Book is allowed.
- ❖ Missing data may be suitable assumed.

- 1A. What is meant by kindling point? Explain with a suitable example. (02)
- 1B. With neat sketches, explain the effect of the engine variables on the flame propagation phase in spark ignition engines. (05)
- 1C. Asphalt pavements on hot summer days exhibit surface temperatures of approximately 65°C. Assuming the surface to be a perfect emitter, calculate the emitted radiant energy per unit surface area. Take the ambient temperature as 20°C. (03)
- 2A. Under what conditions will the Laplace equation of heat conduction become applicable? (02)
- 2B. A small electric heating application uses wire of 3 mm diameter with 0.7 mm thick insulation ($k=0.16 \text{ W/m}^\circ\text{C}$). The heat transfer coefficient on the insulated surface is $40 \text{ W/m}^2\text{C}$. Determine the critical thickness of insulation for the wire. (03)
- 2C. Briefly explain the droplet theory in compression ignition engines with neat sketches. (05)
- 3A. A household electric iron has a cast iron base weighing 1 kg. The surface area of the base is 0.02 m^2 . The iron after usage is switched off and dissipates heat from the base by convection into the ambient air at 20°C. During switching off, the iron base is at a uniform temperature of 130°C. Calculate the temperature of the base after 2 minutes. The convection coefficient between the flat base and the air can be taken as $40 \text{ W/(m}^2\text{.}^\circ\text{C)}$. (04)

- 3B.** Derive an expression for the thermal resistance due to radiation heat transfer between a non-black surface maintained at ' T_1 '°C and ambient environment at ' T_2 '°C. ($T_1 > T_2$). **(03)**
- 3C.** Derive an expression for the logarithmic mean area for hollow cylinders. **(03)**
- 4A.** Calculate the efficiency and effectiveness of a cylindrical fin of diameter 1 cm and 0.6 m long. Its base is maintained at 150°C and is exposed to ambient air at 20°C. Assume the fin material to be copper and let the convective heat transfer coefficient with the ambient air be 30W/(m²°C). **(03)**
- 4B.** Show by dimensional analysis for forced convection, $Nu = \phi(Re, Pr)$. **(05)**
- 4C.** What is meant by pool boiling in liquids? **(02)**
- 5A.** A Cylinder of 300 mm diameter and 1.6 m height is kept horizontally at 36.5°C. Surrounding air is at a temperature of 13.5°C. Find the amount of heat lost by the cylinder surface in kJ/hr. **(04)**
- 5B.** State and prove the Kirchoff's law of radiation. **(03)**
- 5C.** Derive an expression for the efficiency of a fin having an adiabatic free tip. **(03)**
- 6A.** Using Nusselt's method, obtain an expression for the film thickness of the condensate for film-wise condensation across a vertical plate. **(06)**
- 6B.** With neat sketch, explain the working of parallel flow heat exchangers. **(04)**