

Exam Date & Time: 25-Jul-2022 (09:00 AM - 12:00 PM)



MANIPAL ACADEMY OF HIGHER EDUCATION

VI SEMESTER B.TECH MAKE UP EXAMINATIONS, JULY 2022

DESIGN AND DRAWING OF CHEMICAL PROCESS EQUIPMENT [CHE 3251]

Marks: 50

Duration: 180 mins.

A

Answer all the questions.

Section Duration: 180 mins

Instructions to Candidates: Answer ALL questions Missing data may be suitably assumed

- 1) A cylindrical pressure vessel 1.8 m outer diameter and 5 m in height is subjected to an internal pressure of 8 kgf/cm^2 . Corrosion allowance is 2 mm. What will be the minimum thickness of the vessel if the vessel is fabricated as Class A vessel ($J = 1$), Class B vessel ($J = 0.85$), Class C vessel ($J = 0.5$ & $J = 0.7$). Take allowable stress of the material = 1400 kgf/cm^2 . (5)
 - A)
 - B)
- 2) A pressure vessel has inside diameter 1470 mm and a plate thickness of 5mm. Inside diameter of flange is 1482 mm. A gasket is provided over the flange face. Gasket factor is 2.0 and the gasket seating stress is 120 kgf/cm^2 . Inside diameter of gasket is 1485 mm. (5)
 - A) Pressure inside the vessel is 2.5 kgf/cm^2 . Permissible stress of flange material is 1060 kgf/cm^2 . Permissible stress in bolts under atmospheric condition is 600 kgf/cm^2 . Permissible stress in bolts at operating conditions is 550 kgf/cm^2 . Diameter of bolt is 20 mm. Check whether the gasket width is sufficient to keep it away from crushing out. (5)
 - B) While doing thermal design of a 1-2 STHE you end up with the following results: Length of tubes = 5m; No. of tubes = 400; Shell ID = 600 mm; $\Delta P_t = 65 \text{ kN/m}^2$ & $\Delta P_s = 10 \text{ kN/m}^2$; Available $U_{OD} = 500 \text{ W/m}^2\text{K}$; Required $U_{OD} = 2000 \text{ W/m}^2\text{K}$. Do you feel that this design is satisfactory? If not, what are the strategies will you incorporate to satisfy the design criteria? (5)
- 3) 1-1 STHE is used to heat an organic liquid, which passes through the tubes (30 steel tubes; 5cm ID; 5mm thick) of the exchanger. The density, viscosity, heat capacity and the thermal conductivity of the organic liquid are the same as water (1 g/cc ; 1 cP ; 4.2 kJ/kg K and 0.54 W/m K respectively). The shell-side fluid is water that enters with the maximum linear shell-side fluid velocity of 0.15 m/s . determine the shell side heat transfer coefficient. (5)
 - A) Shell side: (Take Diameter = OD of tubes): $Nu = 0.26 Re_{\max}^{0.6} Pr^{0.33}$; $G_{\max} = (v_{\max} * \rho)$
 - B) 1-1 STHE is used to heat an organic liquid ($50 \text{ m}^3/\text{h}$) from 10°C to 28°C , which passes (5)

through the tubes (30 steel tubes; 5cm ID; 5mm thick) of the exchanger. The density, viscosity, heat capacity and the thermal conductivity of the organic liquid are the same as water (1 g/cc; 1cP; 4.2 kJ/kg K and 0.54 W/ m K respectively). The shell-side fluid is water that enters with flow rate 3×10^4 kg/h and temperature 60°C .

Determine the tube side heat transfer coefficient. If the overall heat transfer coefficient is $500 \text{ W/m}^2\text{K}$, determine the length of the tubes.

- 4) A water-cooled, 1-1 shell-and-tube freon condenser with in-tube condensation of R-22 @ 37°C has to be designed. City water (Inlet and outlet temperatures are 18°C & 26°C respectively) is used as coolant. The physical properties at the average temperature of the coolant are: $c_{pL} = 4.181 \text{ kJ/kg K}$; $\text{viscosity}_{(L)} = 959 \times 10^{-6} \text{ Pa.s}$; $k_L = 0.606 \text{ W/m K}$; $\text{Pr} = 6.61$.
A) Determine the
- Shell side heat transfer coefficient
 - Overall heat transfer coefficient if the tube side heat transfer coefficient is $157 \text{ W/m}^2\text{K}$.

Design parameters:

Design cooling load: 125 kW; One tube pass; Pitch: 1" Square; Shell dia: 15.25"; Baffle Spacing: 35 cm; Number of Tubes: 137; Size of tubes: 0.75" OD & 0.68" ID; Vapor quality = 50%.

- B) A water-cooled, 1-1 shell-and-tube freon condenser with in-tube condensation of R-22 @ 37°C ($c_{pL} = 1.305 \text{ kJ/kg K}$; $v_L = 8.3734 \times 10^{-4} \text{ m}^3/\text{kg}$; $v_g = 0.01643 \text{ m}^3/\text{kg}$; $\text{viscosity}_{(L)} = 1.86 \times 10^{-4} \text{ Pa.s}$; $\text{viscosity}_{(g)} = 1.39 \times 10^{-5} \text{ Pa.s}$; $k_L = 0.082 \text{ W/m K}$; latent heat = 169 kJ/kg ; $\text{Pr} = 2.96$) has to be designed. City water (Inlet and outlet temperatures are 18°C & 26°C respectively) is used as coolant. Determine the
- Tube side heat transfer coefficient using Shaw theory.
 - Length of the condenser if the overall heat transfer coefficient is $142 \text{ W/m}^2\text{K}$.

Design parameters:

Design cooling load: 125 kW; One tube pass; Pitch: 1" Square; Shell dia: 15.25"; Baffle Spacing: 35 cm; Number of Tubes: 137; Size of tubes: 0.75" OD & 0.68" ID; Vapor quality = 50%.

- 5) A forward-feed evaporator (triple effect) is used to evaporate a solution. Saturated steam at 8.5 atm abs is being used. The feed rate enters at 16°C .
Take
A) $\Delta T_1 = 28 \text{ K}$, $\Delta T_2 = 23 \text{ K}$ and $\Delta T_3 = 42 \text{ K}$.
- Determine enthalpy of steam, vapor and liquid streams of each evaporator:
- B) A forward-feed evaporator (triple effect) is being used to evaporate a solution containing 10 wt% solids to a concentrated solution of 50 wt%. The feed rate is 22680 kg/h at 26.7°C . Using the following enthalpy values, calculate the steam and liquid flow rates of each evaporator.

Enthalpies of each stream		
Steam Enthalpies, kJ/kg	Vapor Enthalpies, kJ/kg	Liquid Enthalpies, kJ/kg
		@ T_F , $h_F = 111.95$
@ T_{s1} , $\lambda_{s1} = 2199.35$	@ T_1 , $H_1 = 2684$	@ T_1 , $h_{L1} = 440.21$
@ T_1 , $\lambda_{s2} = 2243.18$	@ T_2 , $H_2 = 2654.62$	@ T_2 , $h_{L2} = 364.35$
@ T_2 , $\lambda_{s3} = 2290.27$	@ T_3 , $H_3 = 2600.9$	@ T_3 , $h_{L3} = 230.24$

Useful Energy Balance Equations:

- $F(h_F - H_1) + S\lambda_{s1} = L_1(h_{L1} - H_1)$
- $L_1(h_{L1} - \lambda_{s2} - H_2) + F\lambda_{s2} = L_2(h_{L2} - H_2)$
- $L_2(h_{L2} - \lambda_{s3} - H_3) + L_1\lambda_{s3} = L_3(h_{L3} - H_3)$

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