

Exam Date &amp; Time: 14-May-2022 (10:00 AM - 01:00 PM)



**MANIPAL INSTITUTE OF TECHNOLOGY**  
 MANIPAL  
 (A constituent unit of MAHE, Manipal)

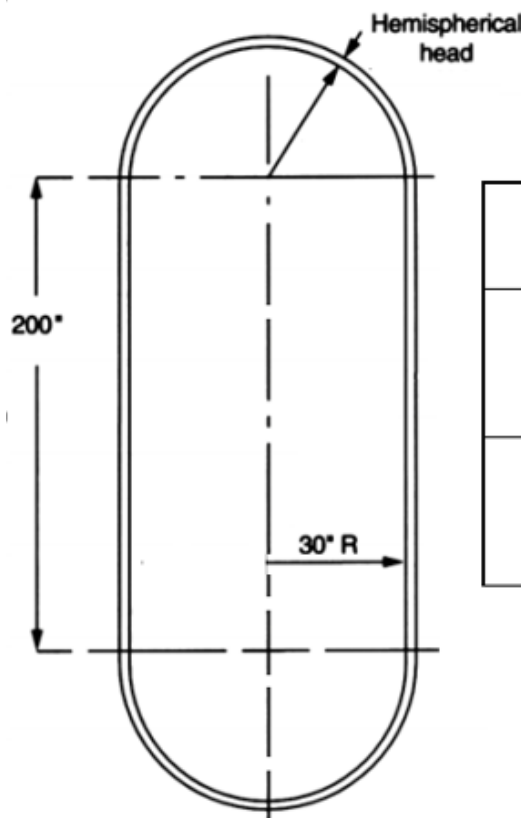
VI SEMESTER B.TECH END SEMESTER EXAMINATIONS, MAY 2022

**DESIGN AND DRAWING OF CHEMICAL PROCESS EQUIPMENT [CHE 3251]****Marks: 50****Duration: 180 mins.****A****Answer all the questions.**

Instructions to Candidates: Answer ALL questions Missing data may be suitably assumed

- 1) Determine the minimum wall thickness of the shell of 200" length (Grade 70) and head of 30" radius (Grade I) from the figure to withstand a pressure of 640 psi (absolute). The reactants are at a temperature of 640°F. Assume that full radiography is done for the welded joints. (Take,  $1 \text{ psi} = 6.9 \times 10^{-3} \text{ N/mm}^2$ )

A)



Material	Temperature limit (°F)	Allowable stress (psi)
Grade 70	640	17500
	704	16600
	800	14500
Grade I	640	18400
	704	16600
	800	12000

(5)

- B) A vacuum distillation column is to operate under a pressure of 50 mmHg. The column diameter is 1 m and the plate spacing 0.5 m. The material of construction is carbon steel and the maximum operating temperature 50 °C. If the vessel thickness is 10 mm, check if this is sufficient or not. If it is insufficient find out the required thickness. (5)
- 2) A vessel is designed for an internal pressure 100 kg<sub>f</sub>/cm<sup>2</sup>. A spiral wound metal, fibre stainless steel gasket with inside diameter 36 cm and width 2.0 cm is used. Gasket factor is 3.0, while gasket seating stress is 675 kg<sub>f</sub>/cm<sup>2</sup>. Permissible stress for bolt material at atmospheric and operating condition is 1300 kg<sub>f</sub>/cm<sup>2</sup>. Check whether the gasket width is sufficient to keep it away from crushing out? (5)
- A)
- B) Detail the ways to modify the internal heat transfer coefficient in the process design of a shell and (5)

tube heat exchanger:

- 3) 1-1 STHE is used to heat an organic liquid, which passes through the tubes (30 steel tubes; 5cm ID; 5mm thick) of the exchanger. The density, viscosity, heat capacity and the thermal conductivity of the organic liquid are the same as water (1 g/cc; 1cP; 4.2 kJ/kg K and 0.54 W/ m K respectively).  
 A) The shell-side fluid is water that enters with the maximum linear shell-side fluid velocity of 0.15 m/s. Determine the shell side heat transfer coefficient. (5)  
 Use,  $Nu = 0.26 Re_{max}^{0.6} Pr^{0.33}$ ;  $G_{max} = (v_{max} * \rho)$   
 In the above equation, take Diameter = OD of tubes
- B) 1-1 STHE is used to heat an organic liquid (50m<sup>3</sup>/h) from 10°C to 28°C, which passes through the tubes (30 steel tubes; 5cm ID; 5mm thick) of the exchanger. The density, viscosity, heat capacity and the thermal conductivity of the organic liquid are the same as water (1 g/cc; 1cP; 4.2 kJ/kg K and 0.54 W/ m K respectively). The shell-side fluid is water that enters with flow rate 3x10<sup>4</sup> kg/h and temperature 60°C. (5)
- Determine the tube side heat transfer coefficient.
  - If the overall heat transfer coefficient is 500 W/m<sup>2</sup>K, determine the length of the tubes.
- 4) A water-cooled, 1-1 shell-and-tube freon condenser with in-tube condensation of R-22 @37°C has to be designed. City water (Inlet and outlet temperatures are 18°C & 26°C respectively) is used as coolant. The physical properties at the average temperature of the coolant are:  $c_{pL} = 4.181$  kJ/kg K;  $\mu_L = 959 \times 10^{-6}$  Pa.s;  $k_L = 0.606$  W/m K;  $Pr = 6.61$ . Fouling resistance:  $1.76 \times 10^{-4}$  m<sup>2</sup> K/W for both inside and outside. Determine the (5)
- Shell side heat transfer coefficient
  - Overall heat transfer coefficient if the tube side heat transfer coefficient is 300 W/ m<sup>2</sup>K.

Design parameters:

Design cooling load: 100 kW; One tube pass; Pitch: 1" Square; Shell dia: 15.25"; Baffle Spacing: 35 cm; Number of Tubes: 137; Size of tubes: 0.75" OD & 0.68" ID; Vapor quality = 50%.

- B) A water-cooled, 1-1 shell-and-tube freon condenser with in-tube condensation of R-22 @37°C ( $c_{pL} = 1.305$  kJ/kg K;  $v_L = 8.3734 \times 10^{-4}$  m<sup>3</sup>/kg;  $v_g = 0.01643$  m<sup>3</sup>/kg;  $\mu_L = 1.86 \times 10^{-4}$  Pa.s;  $\mu_g = 1.39 \times 10^{-5}$  Pa.s;  $k_L = 0.082$  W/m K; latent heat = 169 kJ/kg;  $Pr = 2.96$ ) has to be designed. City water (Inlet and outlet temperatures are 18°C & 26°C respectively) is used as coolant. Determine the (5)
- Tube side heat transfer coefficient using *Cavallini and Zecchin* theory.
  - Length of the condenser if the overall heat transfer coefficient is 201 W/ m<sup>2</sup>K.

Design parameters:

Design cooling load: 100 kW; One tube pass; Pitch: 1" Square; Shell dia: 15.25"; Baffle Spacing: 35 cm; Number of Tubes: 137; Size of tubes: 0.75" OD & 0.68" ID; Vapor quality = 50%.

- 5) A forward-feed evaporator (triple effect) is used to evaporate a solution. Saturated steam at 205.5kPa abs is being used. The feed enters at 26.7 °C.  
 Take (5)  
 $\Delta T_1 = 16$  K,  $\Delta T_2 = 18$  K and  $\Delta T_3 = 32$  K.

Determine enthalpy of steam, vapor and liquid streams of each evaporator:

- B) A forward-feed evaporator (triple effect) is being used to evaporate a solution containing 5 wt% solids to a concentrated solution of 50 wt%. The feed rate is  $2 \times 10^5$  kg/h at 16°C. Using the (5)

following enthalpy values, calculate the steam and liquid flow rates of each evaporator.



Enthalpies of each stream		
Steam Enthalpies, kJ/kg	Vapor Enthalpies, kJ/kg	Liquid Enthalpies, kJ/kg
		@ $T_F$ , $h_F = 67.1$
@ $T_{s1}$ , $\lambda_{s1} = 2038$	@ $T_1$ , $H_1 = 2741$	@ $T_1$ , $h_{L1} = 615$
@ $T_1$ , $\lambda_{s2} = 2125$	@ $T_2$ , $H_2 = 2708$	@ $T_2$ , $h_{L2} = 512$
@ $T_2$ , $\lambda_{s3} = 2196$	@ $T_3$ , $H_3 = 2644$	@ $T_3$ , $h_{L3} = 335$

Useful Energy Balance Equations:

- $F(h_F - H_1) + S\lambda_{s1} = L_1(h_{L1} - H_1)$
- $L_1(h_{L1} - \lambda_{s2} - H_2) + F\lambda_{s2} = L_2(h_{L2} - H_2)$
- $L_2(h_{L2} - \lambda_{s3} - H_3) + L_1\lambda_{s3} = L_3(h_{L3} - H_3)$

-----End-----