



DEPARTMENT OF MECHATRONICS

VI SEMESTER B.TECH. MECHATRONICS

END SEMESTER EXAMINATION, MAY 2023

SUBJECT: **ENERGY AND HEAT TRANSFER [MTE 3252]**

Time: 3 hours

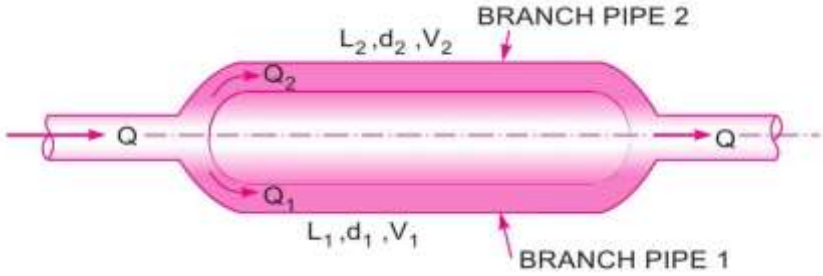
Max. Marks: 50

Instructions to Candidates:

- ❖ Answer **ALL** the questions.
- ❖ Missing data may be suitably assumed and justified.

Q. No	PROBLEM STATEMENT	M	CO	PO	LO	BL
1A.	Two Carnot engines A and B are connected in series between two thermal reservoirs maintained at 1000 K and 100 K respectively. Engine A receives 1680 kJ of heat from the high-temperature reservoir and rejects heat to the Carnot engine B. Engine B takes in heat rejected by engine A and rejects heat to the low-temperature reservoir. If engines A and B have equal thermal efficiencies, Determine (a) The heat rejected by engine B (b) The temperature at which heat is rejected by engine, A	4	1	1,3	1,2	3
1B.	Two ends of a copper rod ($k = 380 \text{ W/(m-K)}$), 15 mm diameter and 300 mm long are connected to two walls, each maintained at 300°C . Air is blown across the rod with a heat transfer coefficient of $20 \text{ W/(m}^2\text{K)}$. Air temperature is 40°C . Determine: (i) midpoint temperature of the rod (ii) net heat transfer to atmosphere.	3	3	1,3	1,2	3
1C.	The aluminium square fins ($0.5 \text{ mm} \times 0.5 \text{ mm}$) of 10 mm length are provided on a surface of semiconductor electronic device to carry 1W of energy generated by electronic device. The temperature at the surface of the device should not exceed 80°C when surrounding temperature is 40°C . Taking the following data, Calculate the number of fins required to carry out above duty. Neglect the heat loss from the end of fins k (aluminium) = $200 \text{ W/m}^\circ\text{C}$; $h = 15 \text{ W/m}^2\text{C}$.	3	3	1,3	1,2	3
2A.	A 0.8 m high, 1.5 m wide double pane window consists of two 4 mm thick layers of glass ($k = 78 \text{ W/mK}$) and is separated by a 10 mm wide stagnant air space ($k = 0.026 \text{ W/mK}$). The room is at 20°C and the outside air is at 10°C . The heat transfer coefficients are $h_i = 10$ and $h_o = 40 \text{ W/m}^2\text{K}$. Determine (i) The rate of heat transfer through the window, (ii) the inside surface temperature	4	2	1,3	1,2	3
2B.	Water ($C_p = 4.187 \text{ kJ/kg K}$) is heated at the rate of 1.4 kg/s from 40°C to 70°C by an oil ($C_p = 1.9 \text{ kJ/kg K}$) entering at 110°C and leaving at 60°C in a counter flow heat exchanger. If $U_o = 350 \text{ W/m}^2 \text{ K}$, determine the surface area required.	3	3	1,3	1,2	3

2C.	In a parallel flow double-pipe heat exchanger water flows through the inner pipe and is heated from 20°C to 70°C. Oil flowing through the annulus is cooled from 200°C to 100°C. It is desired to cool the oil to a lower exit temperature by increasing the length of the heat exchanger. Determine the minimum temperature to which the oil may be cooled.	3	3	1,3	1,2	3
3A.	A 4 kg/s product stream from a distillation column is to be cooled by a 3 kg/s water stream in a counter-flow heat exchanger. The hot and cold stream inlet temperatures are 400K and 300K respectively, and the area of the exchanger is 30 m ² . If the overall heat transfer coefficient is estimated to be 820 W/m ² K, determine the product stream outlet temperature, if its specific heat is 2500 J/kg-K and the coolant outlet temperature using NTU Method.	4	3	1,3	1,2	3
3B.	In a pipe of 300 mm diameter and 800 m length an oil of specific gravity 0.8 is flowing at the rate of 0.45 m ³ /s. Compute (i) Head lost due to friction, and (ii) Power required to maintain the flow. Take kinematic viscosity of oil as 0.3 stoke.	3	6	1,3	1,2	3
3C.	In a pipe of diameter 350 mm and length 75 m water is flowing at a velocity of 2.8 m/s. Compute the head lost due to friction using Darcy-Weisbach formula. Assume kinematic viscosity of water as 0.012 stoke.	3	6	1,3	1,2	3
4A.	Derive an expression for the velocity distribution for viscous flow through a circular pipe. Also sketch the velocity distribution and shear stress distribution across a section of the pipe.	5	5	1,3	1,2	4
4B.	Prove that the total head loss when pipes are connected in series sum of head loss in individual pipes.	3	6	1,3	1,2	5
4C.	Oil of absolute viscosity 1.5 poise and density 848.3 kg/m ³ flows through a 30 cm I.D. pipe. If the head loss in 3000 m length of pipe is 20 m, assuming a laminar flow, determine. (i) the velocity, (ii) Reynolds number.	2	6	1,3	1,2	3
5A.	An oil of viscosity 9 poise and specific gravity 0.9 is flowing through a horizontal pipe of 60 mm diameter. If the pressure drop in 100 m length of the pipe is 1800 kN/m ² , Determine: (i) The rate of flow of oil. (ii) The centerline velocity. (iii) The total frictional drag over 100 m length. (vi) The velocity gradient at the pipe wall. (v) The velocity and shear stress at 8 mm from the wall.	5	5	1,3	1,2	3
5B.	A main pipe divides into two parallel pipes which again forms one pipe as shown in Figure Q5B. The length and diameter for the first parallel pipe are 2000 m and 1.0 m respectively, while the length and diameter of 2nd parallel pipe are 2000 m and 0.8 m. Determine the rate of flow in each parallel pipe, if total flow in the main is 3.0 m ³ /s. The co-efficient of friction for each parallel pipe is same and equal to 0.005.	3	6	1,3	1,2	3

	 <p>Figure Q5B</p>					
5C.	Compare between the hydraulic gradient line (H.G.L.) and energy gradient line (E.G.L.) for a flow of fluid in the pipe.	2	5	1	1	5